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### ATERAL HEAT FLOW EFFECTS ON THERMOGRAPHIC SENSITIVITY

Ignacio Perez, Paul Kulowitch NAVAL AIR WARFARE CENTER AIRCRAFT DIVISION, PATUXENT RIVER, MD

THE SECOND JOINT NASA/FAA/DoD CONFERENCE ON AGING AIRCRAFT

August 31 - September 3, 1998

Williamsburg Marriott Hotel Williamsburg VA

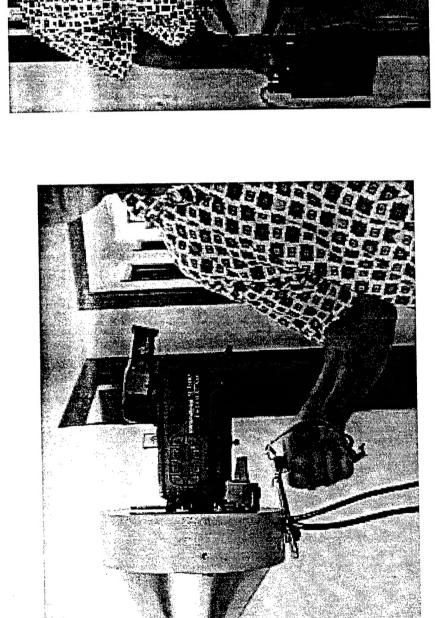
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PUBLIC AFFAIRS OFFICE
NAVAL AIR SYSTEMS COMMAND



# PORTABLE IR CAMERA SYSTEM





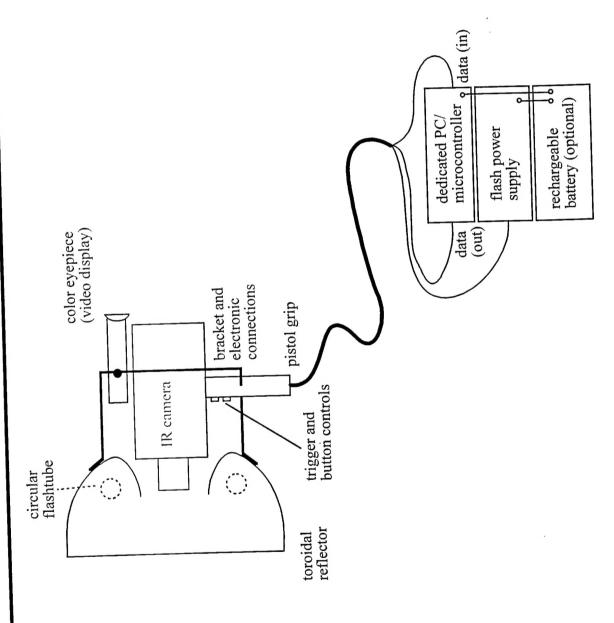
AND POWER SUPPLY

CAMERA HEAD



# PORTABLE IR CAMERA SYSTEM

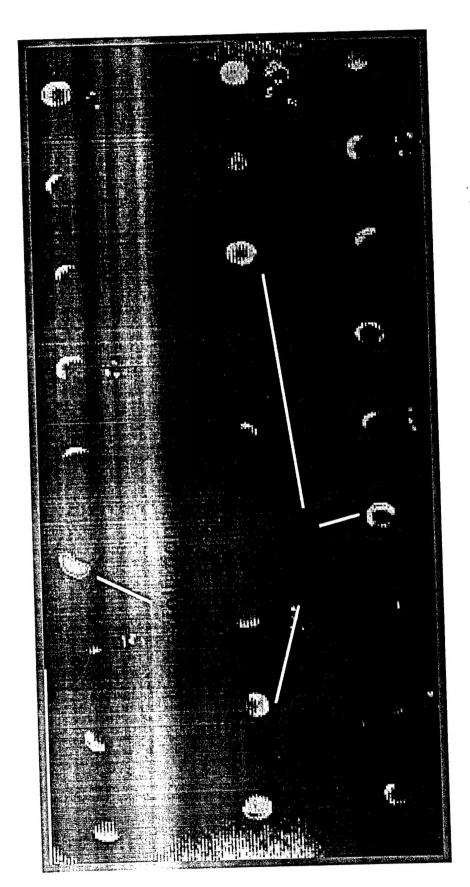






## CORROSION DETECTION





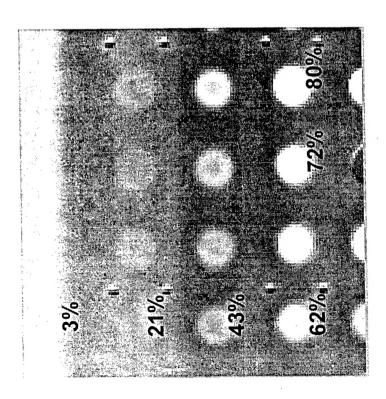


# TEST PANEL & TYPICAL TIME-RESPONSE CURVES

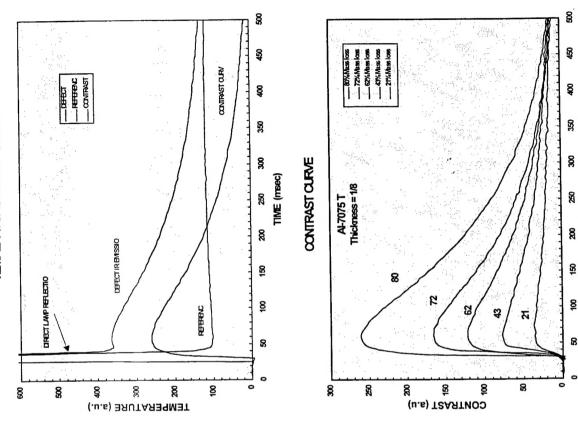


#### TEMPERATURE TIME SEQUENCE





#### 1" Diameter Holes





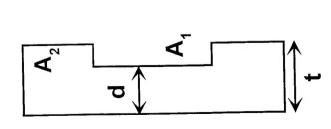
## NO LATERAL HEAT CONDUCTIVITY APPROXIMATION

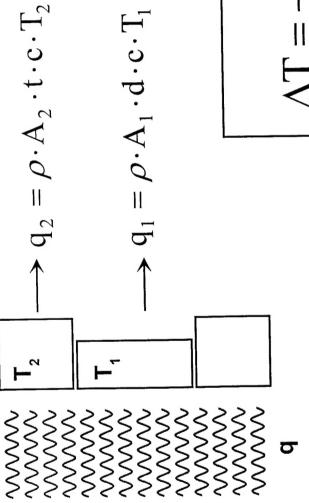


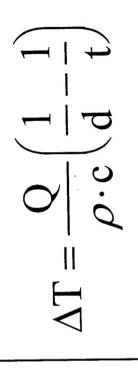
FLAT BOTTOM HOLE

NO LATERAL CONDUCTION APPROXIMATION

 $q = m \cdot c \cdot \Delta T$ 







$$\triangle T = T_1 - T_2$$
  
Q=q/A



### **CONTRAST PROPERTIES**



$$\Delta T = \frac{Q}{\rho \cdot c} \left( \frac{1}{d} - \frac{1}{t} \right)$$

1. THE CONTRAST (AT) INCREASES LINEARLY WITH THE AMOUNT DEPOSITED ENERGY PER UNIT AREA (Q). 2. THE HIGHER THE SPECIFIC HEAT-DENSITY OF A MATERIAL (ρc<sup>↑</sup>) THE SMALLER THE PEAK CONTRAST  $(\Delta T \downarrow)$  3. THE CLOSER THE DEFECT TO THE SURFACE (d ightarrow 0) THE HIGHER THE PEAK CONTRAST ( $\Delta T \rightarrow \infty$ ). 4. AS THE DEFECT DEPTH APPROACHES THE PANEL THICKNESS (d  $\rightarrow$  t) THE CONTRAST VANISHES  $(\Delta T \rightarrow 0)$ . 5. FOR A GIVEN DEFECT DEPTH D, THE THICKER THE PANEL  $(t \to \infty)$  THE LARGER THE CONTRAST ( $\triangle T \rightarrow Q/\rho cd$ ).



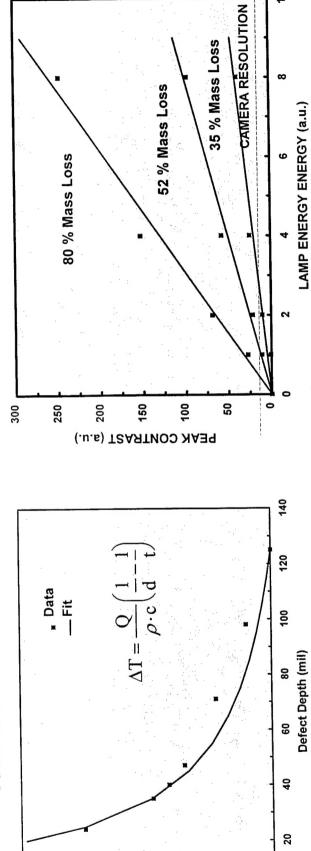
#### SIMPLE MODEL CORRELATION (no lateral heat flow)



Contrast (a.u.)

CONTRAST vs DEPTH

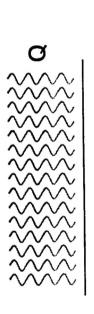
DEPTH OF RESOLUTION VS ENERGY

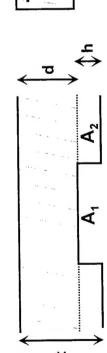




## LATERAL HEAT FLOW MODEL









$$A_1 \cdot d \cdot c \cdot \frac{dT_1}{dt} = k_L \cdot \frac{A_L}{R} (T_2 - T_1)$$

$$\rho \cdot A_2 \cdot d \cdot c \cdot \frac{dT_2}{dt} = k_L \cdot \frac{A_L}{R} (T_1 - T_2) + k \cdot \frac{A_2}{d+h} (T_2' - T_2)$$

$$\rho \cdot A_2 \cdot h \cdot c \cdot \frac{dT_2'}{dt} = k \cdot \frac{A_2}{d+h} (T_2 - T_2')$$

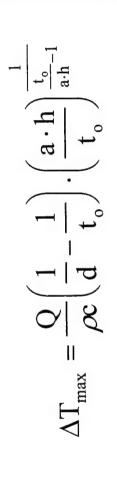


## LATERAL HEAT FLOW EFFECTS



#### $-\frac{1+r}{e^{-\frac{(d+h)}{\omega}t}}$ $-\frac{a}{d(d+h)}\frac{k}{\mathcal{K}}t$ $\rho x \cdot d \cdot (1 - a + r)$ $\Delta T(t) = -$

$$t_{max} = \frac{\rho c}{k} \frac{d \cdot t_o}{1 - a + r} ln \frac{1 + r}{a}$$



$$a = \frac{k_L}{k} \frac{A_L}{A} \frac{d+h}{R}$$



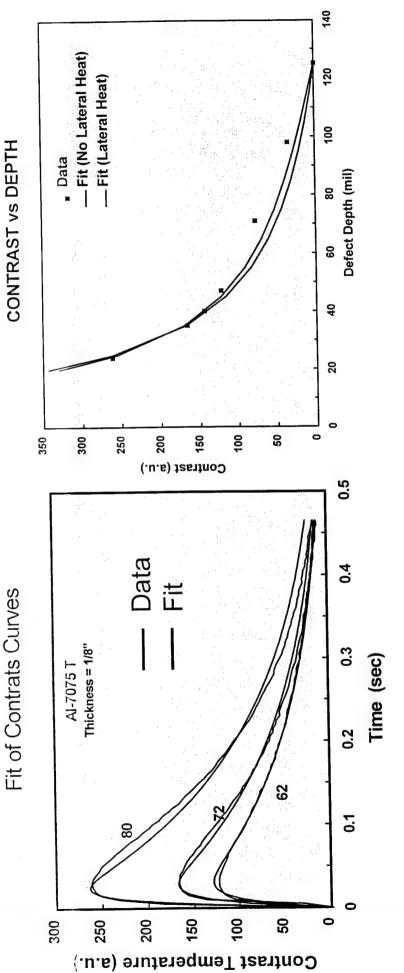
$$r = \frac{d}{h}$$



# THERMAL CONTRAST PREDICATIONS



#### Fit of Contrats Curves



$$\Delta T(t) = \frac{Q}{\rho c \cdot d \cdot (1 - a + r)} \left( e^{-\frac{a}{d(d+h)}\frac{k}{\rho c}t} - e^{-\frac{1 + r}{d(d+h)}\frac{k}{\rho c}t} \right)$$

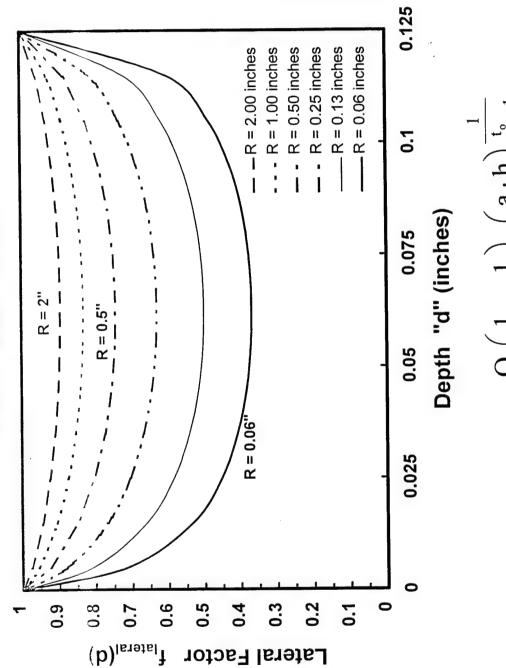
$$\Delta T_{\text{max}} = \frac{Q}{\rho c} \left( \frac{1}{d} - \frac{1}{t_o} \right) \cdot \left( \frac{a \cdot h}{t_o} \right) \frac{\frac{t_o}{t_o} - 1}{t_o}$$



# LATERAL HEAT FACTOR (effective contact conductivity model)

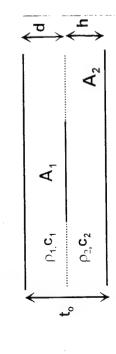


### Lateral Heat Factor



## OTHER MODELING RESULTS





$$\Delta T(t) = \frac{Q}{\rho_1 c_1 \cdot d(r_1 + b \cdot r_1 r_2 - a_1)}.$$

$$\cdot \left(e^{-\frac{a_1 \ k}{Rd \ \rho_1 c_1}t} - e^{-\frac{r_1(1+br_2) \ k}{Rd \ \rho_1 c_1}t}\right)$$

$$\Delta T_{peak} = Q \cdot \left(\frac{1}{d} - \frac{1}{t_o}\right) \frac{\rho_2 c_2}{\rho_1 c_1} \frac{d + h}{(d\rho_1 c_1 + h\rho_2 c_2)}$$
$$\cdot \left[\frac{a_1}{r_1 (1 + b r_2)}\right]^{\frac{1}{1 - \frac{a_1}{r_1 (1 + b r_2)}}}$$

$$a_1 = \frac{k_L}{k} \frac{A_d}{A} \qquad a_2 = \frac{k_L}{k} \frac{A_h}{A}$$

 $\rho_1^{c_1}$  $\rho_2 c_2$ 

р |-

II

d + b

i I

$$(t) = \frac{J}{a \cdot k} \frac{d + h}{(1 + r)(1 + r - a)}.$$

$$\cdot \left\{ (1 + r)(1 - e^{-\frac{a}{d(d + h)}\frac{k}{\rho^c}}) - a(1 - e^{-\frac{1 + r}{d(d + h)}\frac{k}{\rho^c}}) \right\}$$

$$\Delta T(t \to \infty) = \frac{J}{a \cdot k} \cdot h$$

$$a = \frac{k_L}{k} \frac{A_L}{A} \frac{d+h}{R}$$
  $r = \frac{d}{h}$ 

T

$$\Delta T(t) = \frac{Q}{\rho c \cdot d(AkR + A_d k_1 d)} \left( e^{-\frac{b}{d^2} \frac{k}{\rho c}} - e^{-\frac{1}{d^2} \frac{k}{\rho c}} \right)$$

$$\Delta T_{peak} = \frac{Q}{\rho c} \frac{kR(b-1)}{d(dA_d k_1 + RAk)} \cdot \left[b\right]^{\frac{b}{1-b}}$$

$$t_{peak} = \frac{\rho c}{k} \frac{d^2}{b - 1} \ln b$$

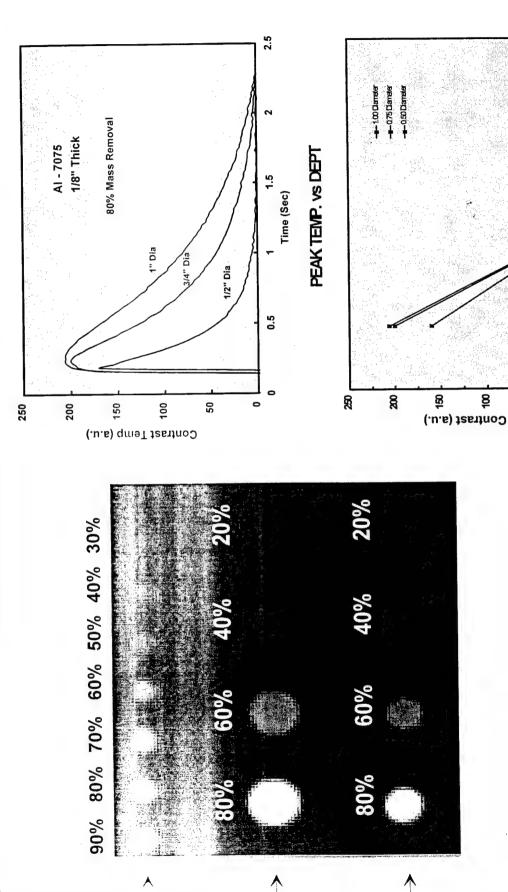
$$b = \frac{k_1}{k} \frac{A_d}{A_1} \frac{d}{R}$$



### **EXPERIMENTAL DATA**

(80% mass removal)





4

8

8

4

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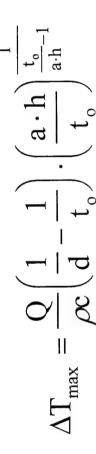
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Depth (mils)

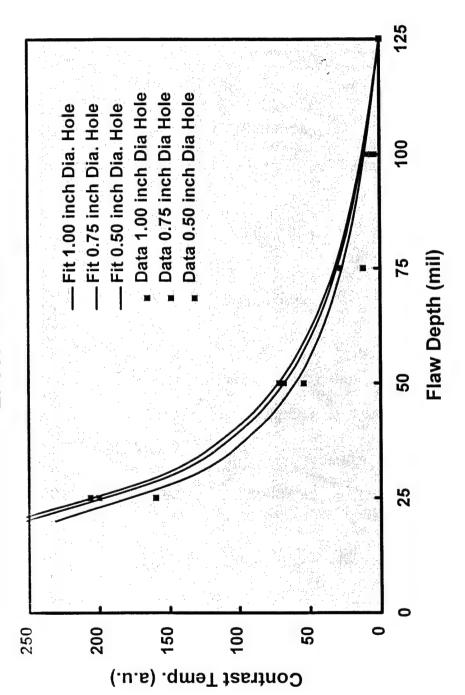


### MODEL CORRELATION (effects of defect size)





Effects of Radii

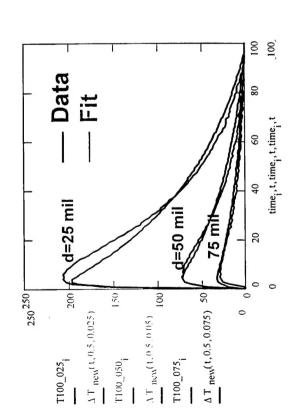




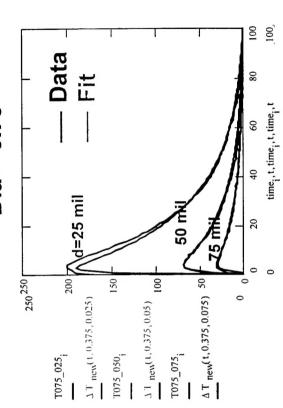
### MODEL TIME-RESPONSE PREDICTIONS (varying defect sizes and locations)



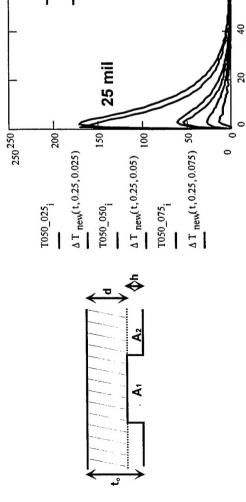




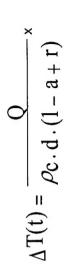
Dia = 0.75"



Dia = 0.50"



time, t, time, t, time, t



Data

世

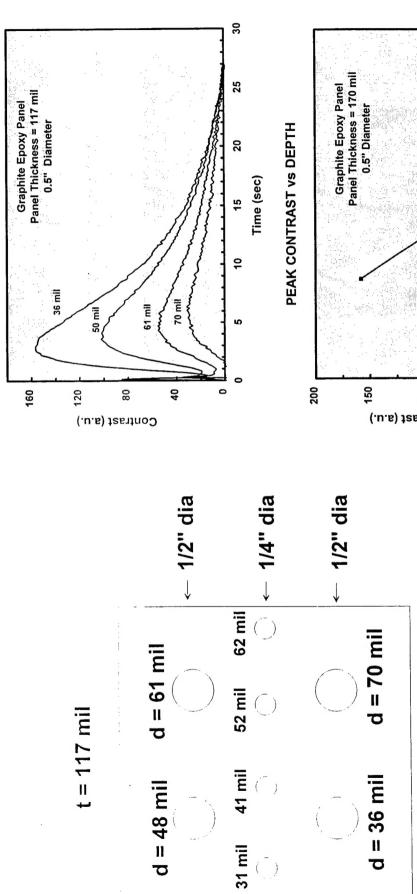
$$\times \left(e^{-\frac{a}{d(d+h)}\frac{k}{\rho c}t} - e^{-\frac{1+r}{d(d+h)}\frac{k}{\rho c}t}\right)$$

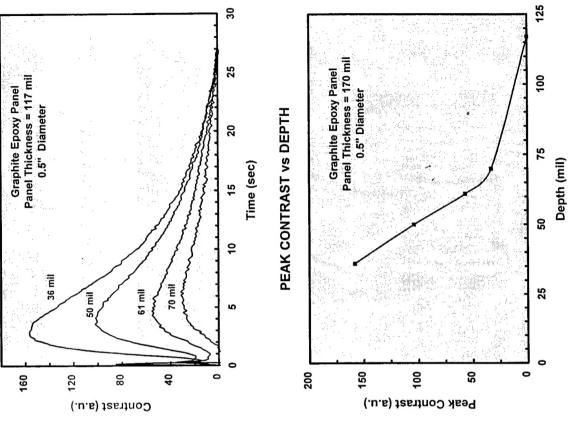


## GRAPHITE EPOXY COMPOSITE PANEL



#### CONTRAST vs TIME

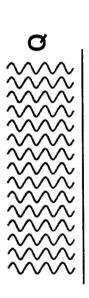






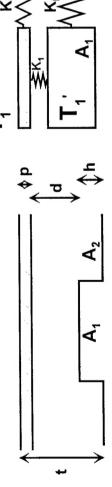
## SIMPLE FINITE ELEMENT APPROXIMATION

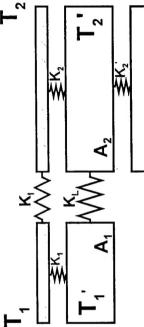




#### Sample

defect





$$\rho \cdot A_1 \cdot p \cdot c \cdot \frac{dT_1}{dt} = k \cdot A_1(T_1' - T_1) + k_L \cdot A_p(T_2 - T_1)$$

$$\rho \cdot A_2 \cdot p \cdot c \cdot \frac{dT_2}{dt} = k \cdot A_2 (T_2' - T_2) + k_L \cdot A_p (T_1 - T_2)$$

$$\rho \cdot A_2 \cdot h \cdot c \cdot \frac{dT_2^{"}}{dt} = k \cdot A_2 (T_2 - T_2^{"})$$



### FITTING RESULTS





